EFFECT OF OVERFLOW NAPPE NON-AERATION ON TSUNAMI BREAKWATER FAILURE

Sanduni Disanayaka Mudiyanselage, MSc Jeremy D. Bricker, PhD, PE Prof. dr. ir. Wim Uijttewaal Dr. ir. Geert Keetels Akshay Patil

Photo: PARI

Delft University of Technology



Outline

- Motivation
 - Kamaishi breakwater during the 2011 Japan tsunami
 - CFD simulation of forces on the Kamaishi breakwater
- Cavity sub-pressure
 - Laboratory experiment
 - Integral conservation of momentum
- Necessary steps





Kamaishi breakwater plan

図4-2 釜石港湾口地区防波堤(南堤)深部2区標準断面図

ケーソン諸元(1)

| | 体積(m ³) | 重量 (tf) |
|---------|---------------------|-----------|
| ケーソン | 4,243.52 | 10,396.62 |
| カウンター材 | 608.92 | 1,400.51 |
| 中 詰 材 | 7,462.57 | 14,178.89 |
| 水張り | 2,480.54 | 2,554.95 |
| 捨コンクリート | 452.85 | 1,041.55 |
| 上部工 | 983.25 | 2,261.48 |
| 合 計 | | 31,834.00 |

ケーソン諸元(2)

| 寸 法(m) | B 26.0(23.0) ×L 30.0 ×H 25.0 | |
|---------|------------------------------|--|
| 重 量(tf) | 11,849(止水蓋 51.4tf 含心) | |
| 喫 水(m) | 16.48m | |

2 工区 設置水深 -22.0m 矩形

| 安 | 定 計 | 算結果 | 波圧時 | 地震時外→内 | 地震時内→外 |
|---------------|--------|--------------|------------|-----------|-----------|
| 7 | - F | 動 | 1.313>1′.2 | (0.926) | (0.926) |
| ŧ. | Б Б | 倒 | 1.735>1.2 | 1.675>1.1 | 1.892>1.1 |
| 端 | P1 (| t /m²) | 66.42<70 | 76.14<80 | 57.62<80 |
| <u>唯</u> 圧 | 作用 | 畐B(m) | 14.57 | 14.76 | 19.50 |
| 支 | н. V | W.L 時 | 1.223>1.0 | | |
| 打 力 | L. V | W.L 時 | 1.244>1.0 | | |
| P | 9形す | べり | 1.666>1.2 | | |



Damage to breakwater

- Caissons displaced toward land
- Rubble mound experienced scour (Arikawa et. al., 2011)
 - From overtopping flow
 - From flow through gap between caissons
- Bearing capacity exceedance (Bricker et al., 2013)

Multibeam SONAR - Tomita et. al., 2012



図-1 東北地方太平洋沖地震津波後の釜石湾口防波堤周辺の 海底地形図

OpenFOAM free surface simulation (Bricker et al., 2013)

Standard k-epsilon turbulence model

- However, experiments (Mitsui et al, 2012) show the overtopping jet does not pull up close to the caisson.
- Possible reason: Continuous eddy viscosity across air-water interface used by VOF allows too much entrainment of air into the impinging jet (an area of strong turbulence).
- Solution: reduce eddy viscosity at interface by neglecting all turbulence in the air phase.

Standard k-epsilon turbulence model in water, no turbulence in air





OpenFOAM simulation result (Bricker et al., 2013)



Nappe types

Development of ΔF



 ΔF will increase the sliding force acting on the caisson affecting the stability



Laboratory physical model



Geometric scale= 1:150



Measured nappe sub-pressure



$$d_a \rho g + P_{atm} = (P_3)_{aerated} + P_{atm}$$

 $d_a \rho_{eff} g + d_{na} \rho_{eff} g + (P_{atm} - \Delta P) = (P_3)_{non-aerated} + P_{atm}$

$$\Delta P = \frac{\rho_{eff}}{\rho} (P_3)_{aerated} - (P_3)_{non-aerated} + d_{na}\rho_{eff}g$$

Theoretical nappe sub-pressure from momentum conservation over integral control volume



x-direction integral momentum equation over control volume

$$\overrightarrow{F_s} + \overrightarrow{F_b} = \frac{d}{dt} \int_{CV} \overrightarrow{u} \rho dV + \int_{CS} \overrightarrow{u} \rho \overrightarrow{u} \cdot \overrightarrow{dA}$$

$$\int_{CS} \overrightarrow{u} \rho \overrightarrow{u} \cdot \overrightarrow{dA} = \rho Q(u_R - uL) \qquad u_R = \frac{Q}{Bd_a} \qquad u_L = \frac{Q}{Bh_1}$$

$$F_{sL} = \int_0^{d_{na}} \rho_{eff} gB(d_{na} - z)dz - \Delta PBH_c + \int_0^{h_1} \rho gB(h_1 - z)dz$$

$$F_{sR} = -\int_0^{d_a} \rho gB(d_a - z)dz$$

$$\Delta P = \frac{\frac{1}{2} \rho gB(h_1^2 - da^2) + \frac{1}{2} \rho_{eff} gBd_{na}^2 - \frac{\rho Q^2}{B} \left(\frac{1}{d_a} - \frac{1}{h_1}\right)}{BH_c}$$

Effect of density ρ_{eff} = 800 kg/m³ in non-aerated case





Non-aerated jet trajectory

Tan (1984) via Chanson (1996) considered vertical component of subpressure-induced acceleration in ballistic trajectory, but here consider both components.



$$a_i = \frac{\Delta P}{\rho d_i}$$

$$\Delta x_{i} = u_{i-1}\Delta t - \frac{1}{2}(a_{i-1}\sin\theta_{i-1})\Delta t^{2} \qquad u_{i}^{2} = u_{i-1}^{2} - 2(a_{i-1}\sin\theta_{i-1})\Delta x_{i}$$

$$\Delta y_{i} = -v_{i-1}\Delta t + \frac{1}{2}\{g + a_{i-1}\cos\theta_{i-1}\}\Delta t^{2} \qquad v_{i}^{2} = v_{i-1}^{2} + 2\{a_{i-1}\cos\theta_{i-1} + g\}\Delta y_{i}$$

$$d_i = \frac{Q}{B_C * \sqrt{u_{i-1}^2 + v_{i-1}^2}}$$
$$\theta_i = \tan^{-1} \frac{\Delta y_i}{\Delta x_i}$$



Non-aerated nappe trajectory





Conclusions

- Cavity sub-pressure affects jet trajectory and forces on caisson
- CFD models have difficulty modeling this due to bubbling and air entrainment
- The water column under a non-aerated nappe may have a lower density than under an aerated nappe due to bubbling



Future work

- Measure cavity sub-pressure directly
- Develop a relation between overflow surcharge and cavity sub-pressure, building on Chanson (1996)
- Extend to unsteady flows
- Investigate what is needed in CFD to get this right (2-D vs. 3-D simulations, turbulence models, bubble physics)





Future Work

- 3D simulation
- Contracted nappe, air enters at sides
- Is that all?

